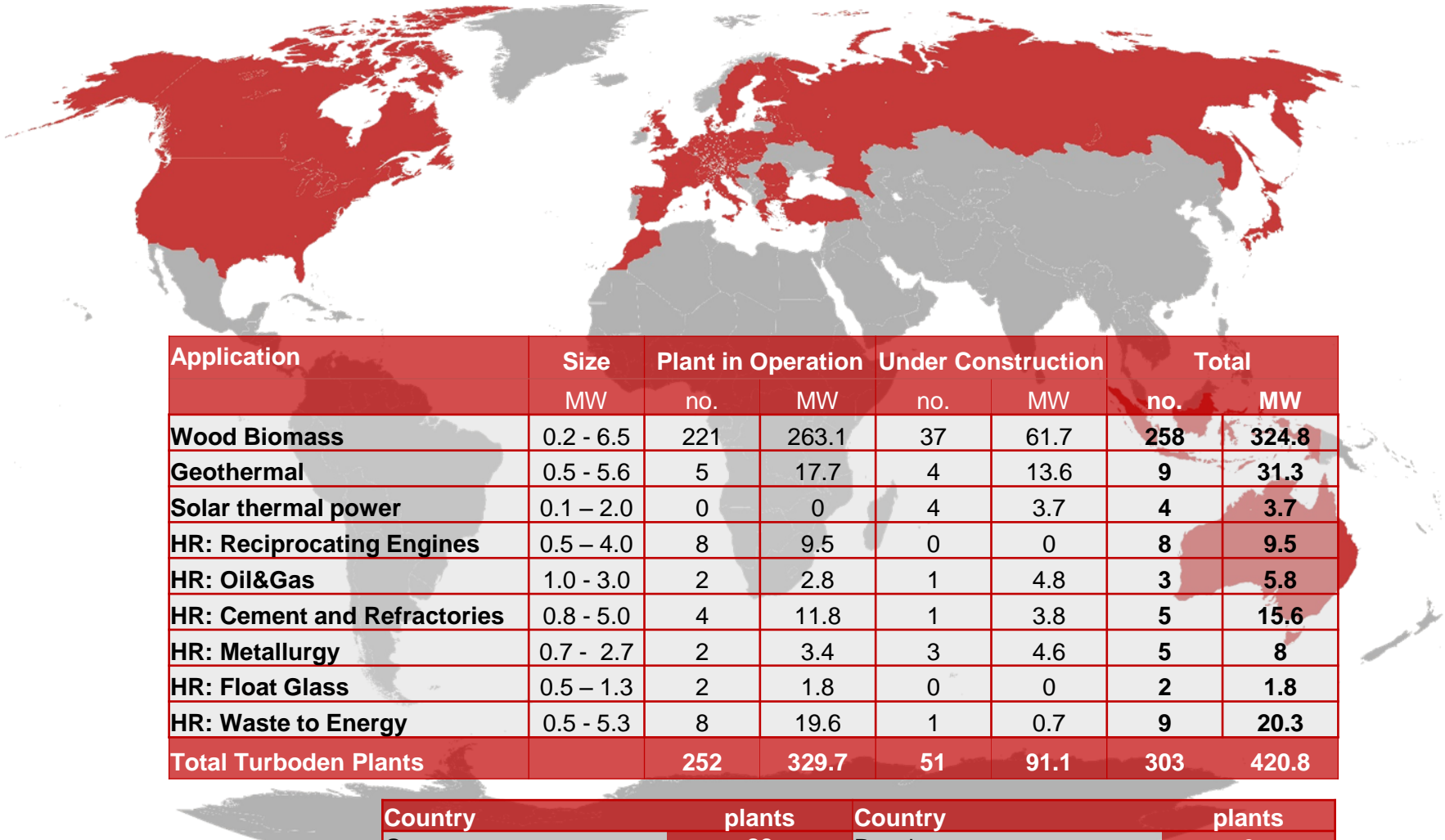


Turboden Solutions in the Oil&Gas Industry

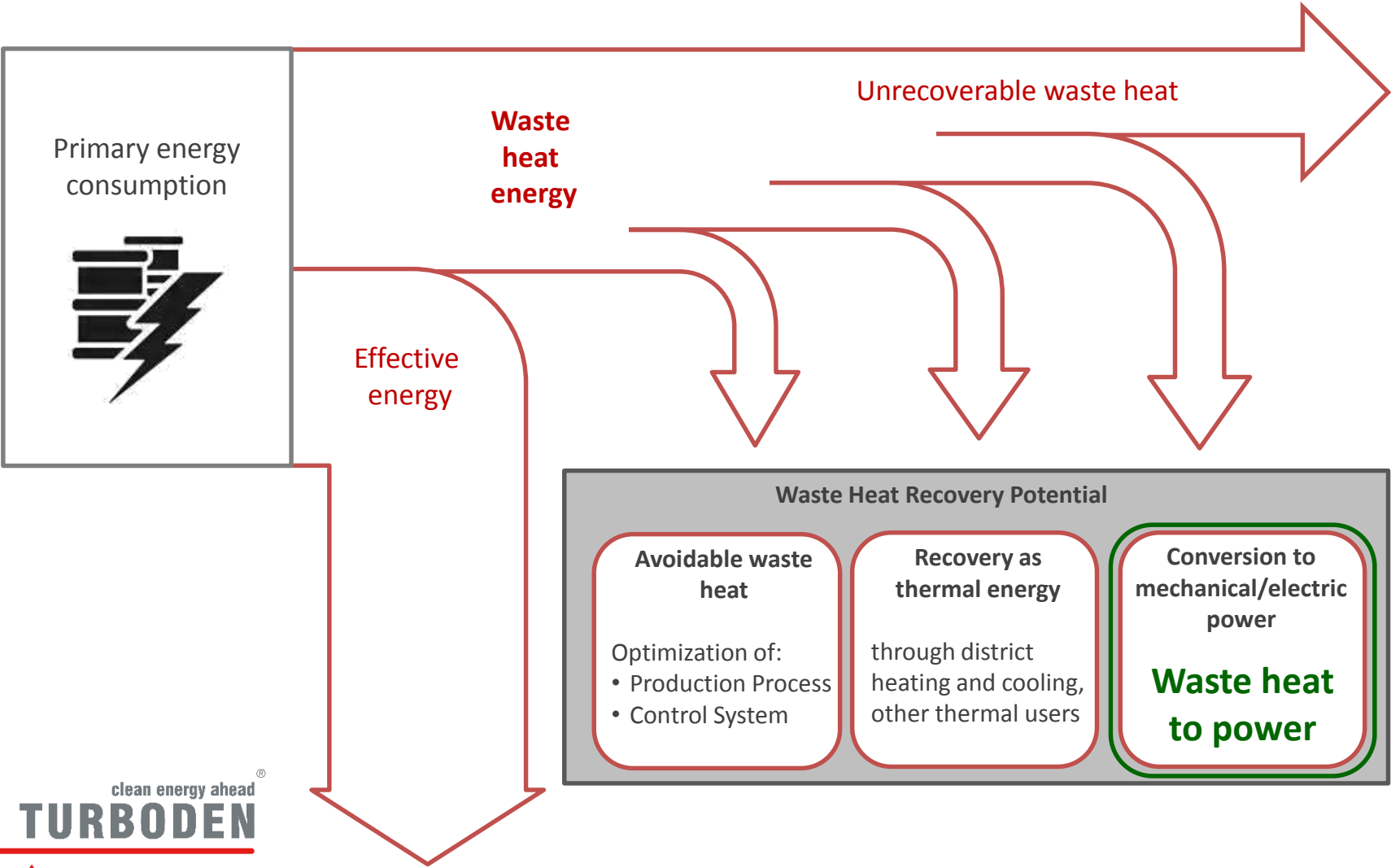
Turboden has currently more than 300 reference plants worldwide



Application	Size	Plant in Operation		Under Construction		Total	
	MW	no.	MW	no.	MW	no.	MW
Wood Biomass	0.2 - 6.5	221	263.1	37	61.7	258	324.8
Geothermal	0.5 - 5.6	5	17.7	4	13.6	9	31.3
Solar thermal power	0.1 – 2.0	0	0	4	3.7	4	3.7
HR: Reciprocating Engines	0.5 – 4.0	8	9.5	0	0	8	9.5
HR: Oil&Gas	1.0 - 3.0	2	2.8	1	4.8	3	5.8
HR: Cement and Refractories	0.8 - 5.0	4	11.8	1	3.8	5	15.6
HR: Metallurgy	0.7 - 2.7	2	3.4	3	4.6	5	8
HR: Float Glass	0.5 – 1.3	2	1.8	0	0	2	1.8
HR: Waste to Energy	0.5 - 5.3	8	19.6	1	0.7	9	20.3
Total Turboden Plants		252	329.7	51	91.1	303	420.8

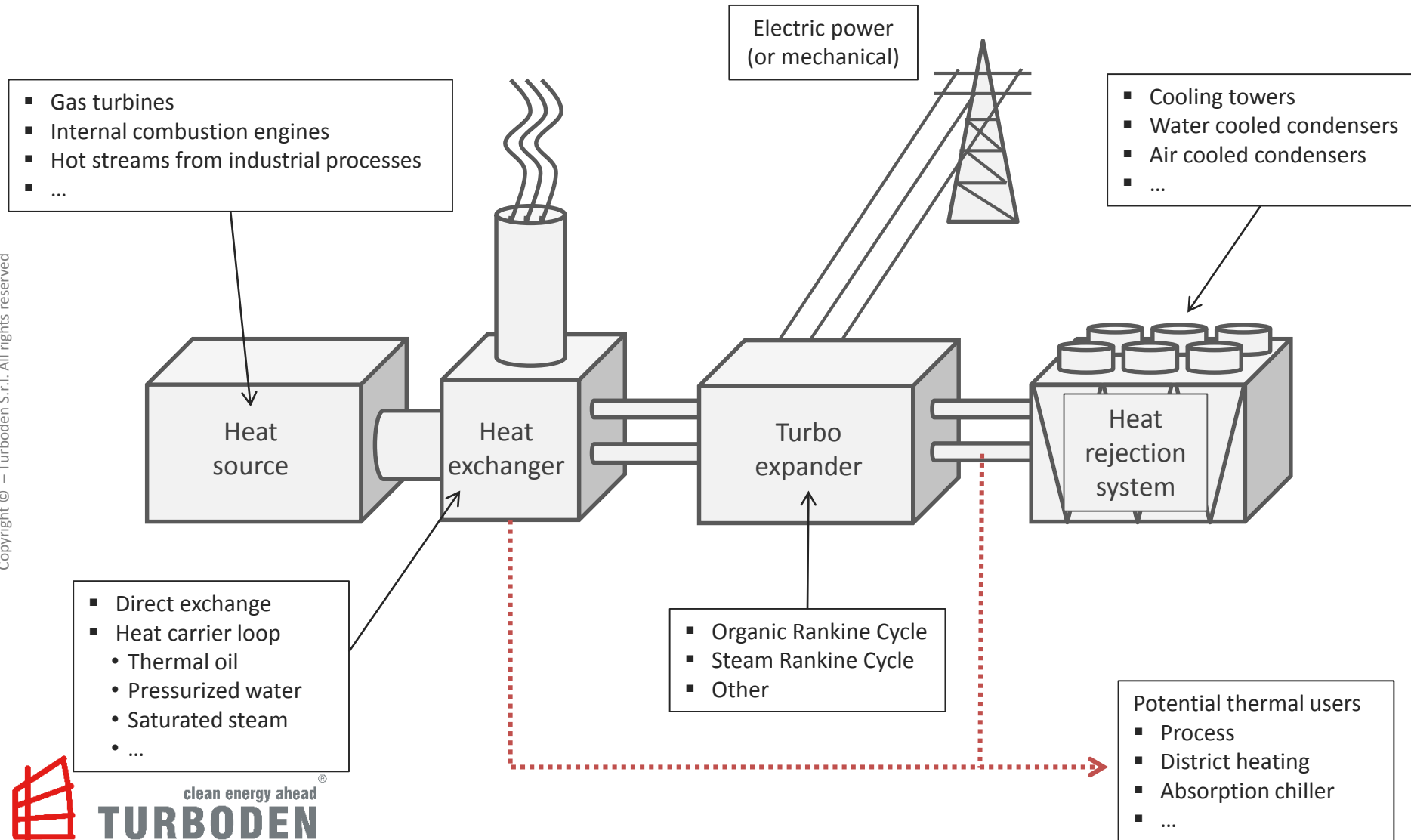
Country	plants	Country	plants
Germany	82	Russia	6
Italy	84	Rest of the World	6
Austria	31	Turkey	5
Rest of Europe	81	North America	8

Waste heat from industrial processes can be recovered and converted into power



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Waste heat to power



What we do



Biomass



Heat recovery



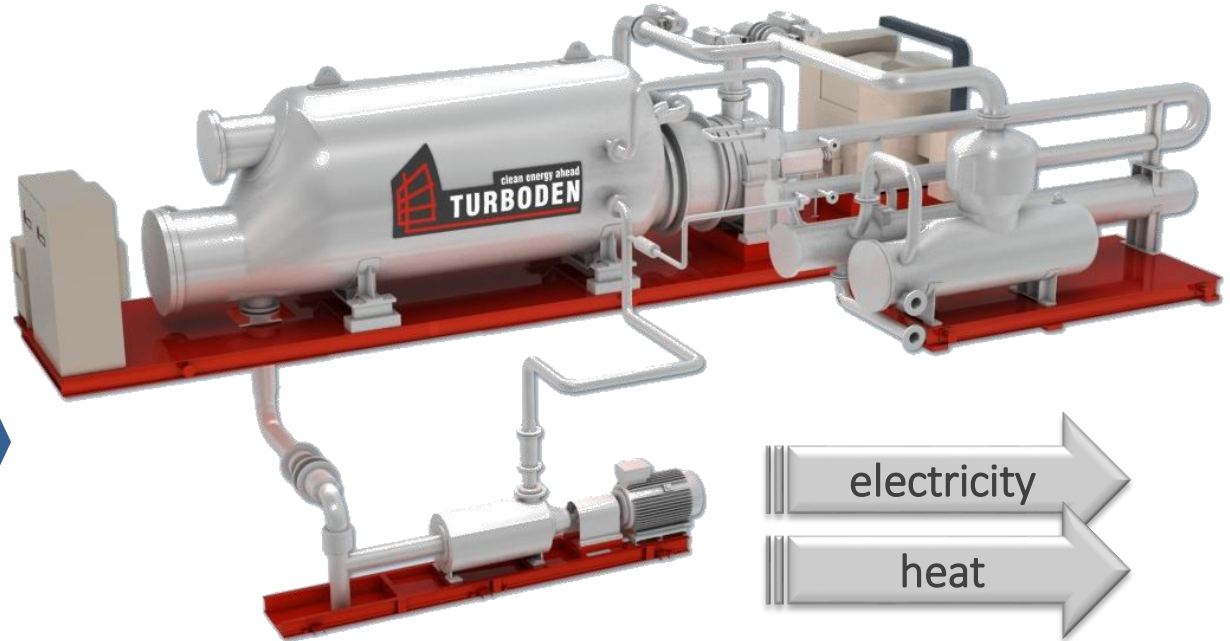
Waste to energy



Geothermal



Solar

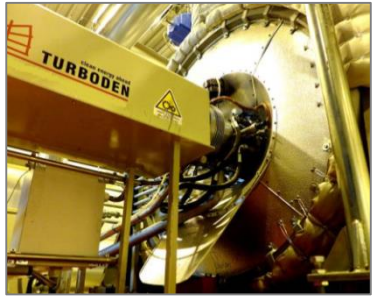


Turboden designs, develops and maintains turbogenerators based on the Organic Rankine Cycle (ORC), a technology for the combined generation of electric power and heat from various renewable sources, particularly suitable for distributed generation.

➤ Turboden solutions from 200 kW to 15 MW electric per single unit



Organic Rankine Cycle: concept



Cycle it is a thermodynamic cycle

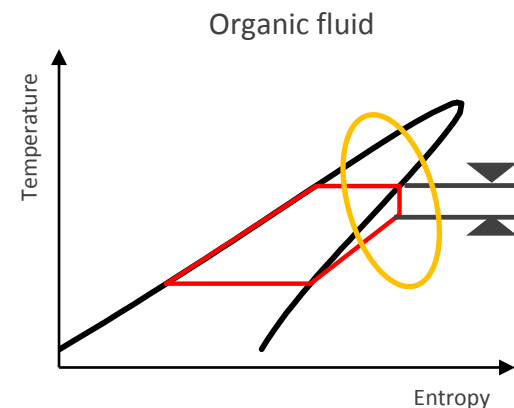
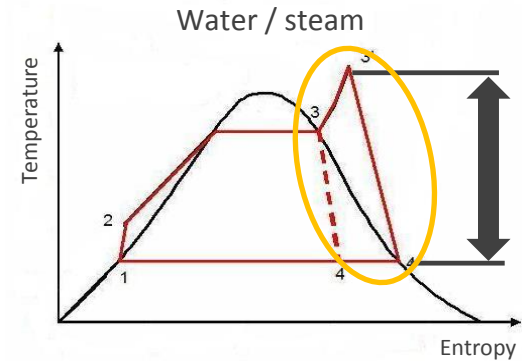
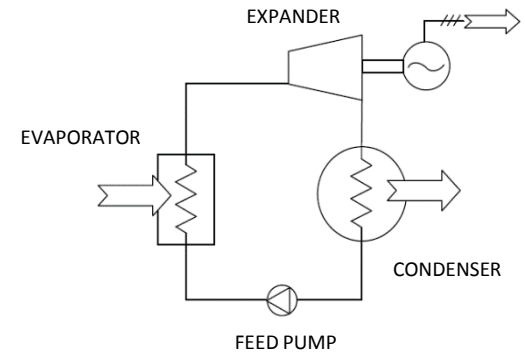
Rankine it is theoretically given by 2 isobar and 2 adiabatic thermodynamic transformations

Organic it exploits an organic working fluid

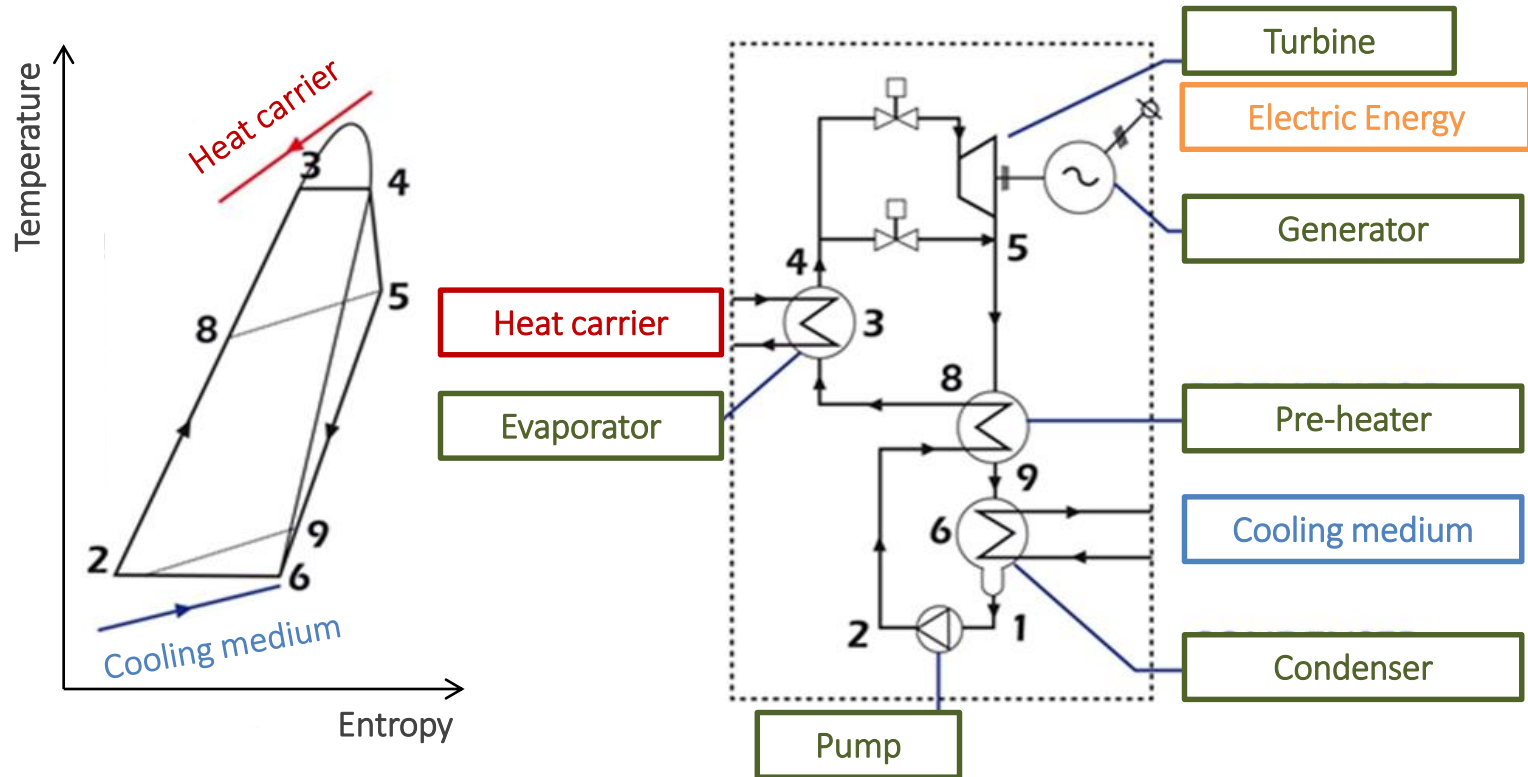


The principle is based on a turbogenerator working as a normal steam turbine to transform thermal energy into mechanical energy and finally into electric energy through an electric generator. **Instead of the water steam**, the ORC system **vaporizes an organic fluid**, characterized by a **molecular mass higher than water**, which leads to a **slower rotation** of the turbine and **to lower pressure and erosion** of the metallic parts and blades.

Efficiency: 98% of incoming thermal power is transformed into **electric power** (around **20%**) and **heat (78%)**, with extremely limited thermal leaks, only 2% due to thermal isolation, radiance and losses in the generator. The electric efficiency obtained in **non-cogeneration** cases is much higher (more than **24%** of the thermal input).

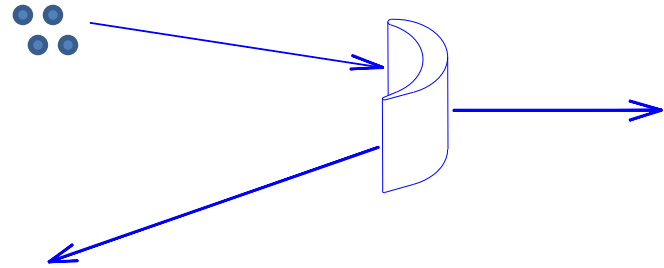


Organic Rankine Cycle: Thermodynamics



The turbogenerator uses the **heat carrier** (e.g. hot temperature thermal oil) to pre-heat and vaporize a suitable organic working fluid in the **evaporator** (8→3→4). The organic fluid vapor powers the **turbine** (4→5), which is directly coupled to the **electric generator** through an elastic coupling. The exhaust vapor flows through the **regenerator** (5→9) where it heats the organic liquid (2→8). The vapor is then condensed in the **condenser** (cooled by the water flow or other) (9→6→1). The organic fluid liquid is finally **pumped** (1→2) to the **regenerator** and then to the evaporator, thus completing the sequence of operations in the closed-loop circuit.

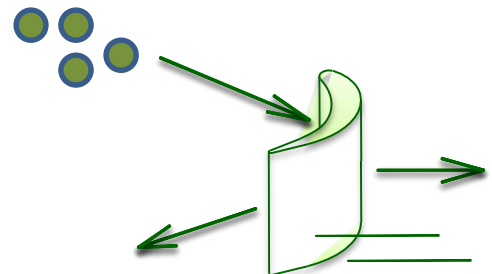
Water vs High Molecular Mass - Working Fluid



The diagram shows a blue water turbine blade with a curved leading edge. Four small blue circles representing water molecules are positioned to the left of the blade. Blue arrows indicate the flow of water from the molecules towards the blade, and a single blue arrow points away from the blade to the right, representing the output flow.

Water

- Small, fast moving molecules
- Metal parts and blade erosion
- Multistage turbine and high speed with mechanical stress



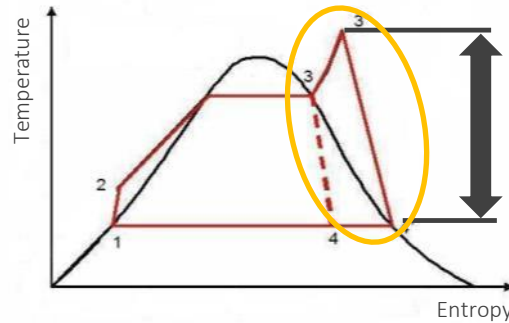
The diagram shows a green high molecular mass fluid turbine blade with a curved leading edge. Four larger green circles representing high molecular mass fluid molecules are positioned to the left of the blade. Green arrows indicate the flow of fluid from the molecules towards the blade, and multiple green arrows point away from the blade to the right, representing the output flow.

High molecular mass fluid

- Large flow rate
- Larger diameter turbine with high efficiency of the turbine (85-90%)
- No wear of blades and metal parts
- Slow rotation speed and few stages (2-6)

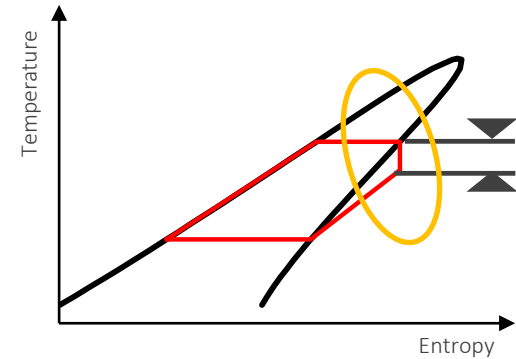
ORC provides significant advantages as compared to steam

Steam Rankine Cycle



- High enthalpy drop
- Superheating needed
- Risk of blade erosion
- Water treatment required
- Highly skilled personnel needed
- High pressures and temperatures in the cycle
- Convenient for large plants and high temperatures
- Low flexibility with significantly lower performances at partial load

Organic Rankine Cycle (ORC)



- Small enthalpy drop
- No need to superheat
- No supercritical pressure
- No risk of blade erosion
- Non-oxidizing working fluid with no corrosion issues
- Minimum personnel and O&M (1)
- Completely automatic (2)
- No blow down
- High flexibility and good performances at partial load
- High availability (average >98%)
- Possibility to work at low temperatures (90+°C)

Thermodynamic features and consequences

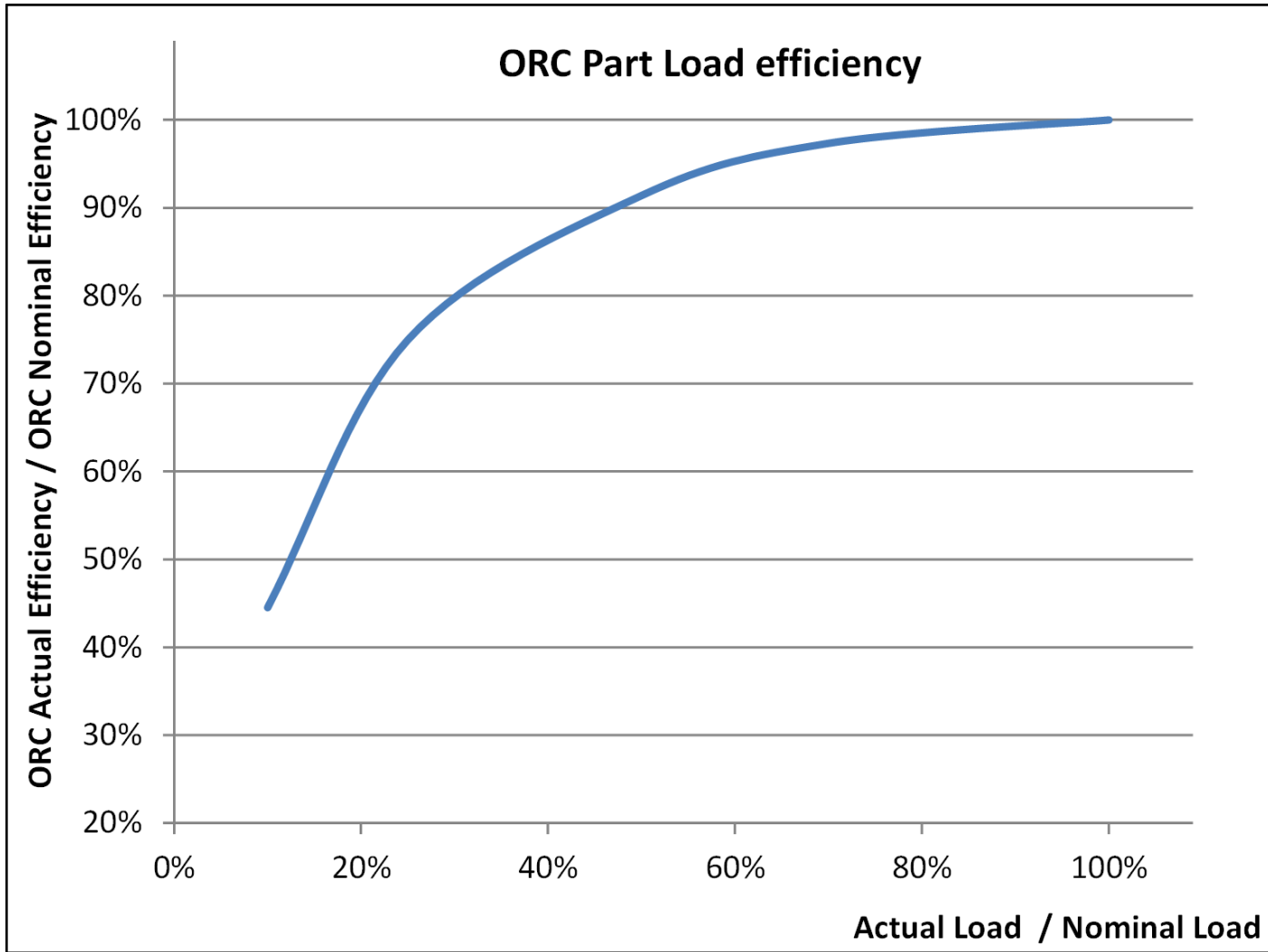
Operation and maintenance costs

Other features

(1) Standard maintenance: 2-3 days per year

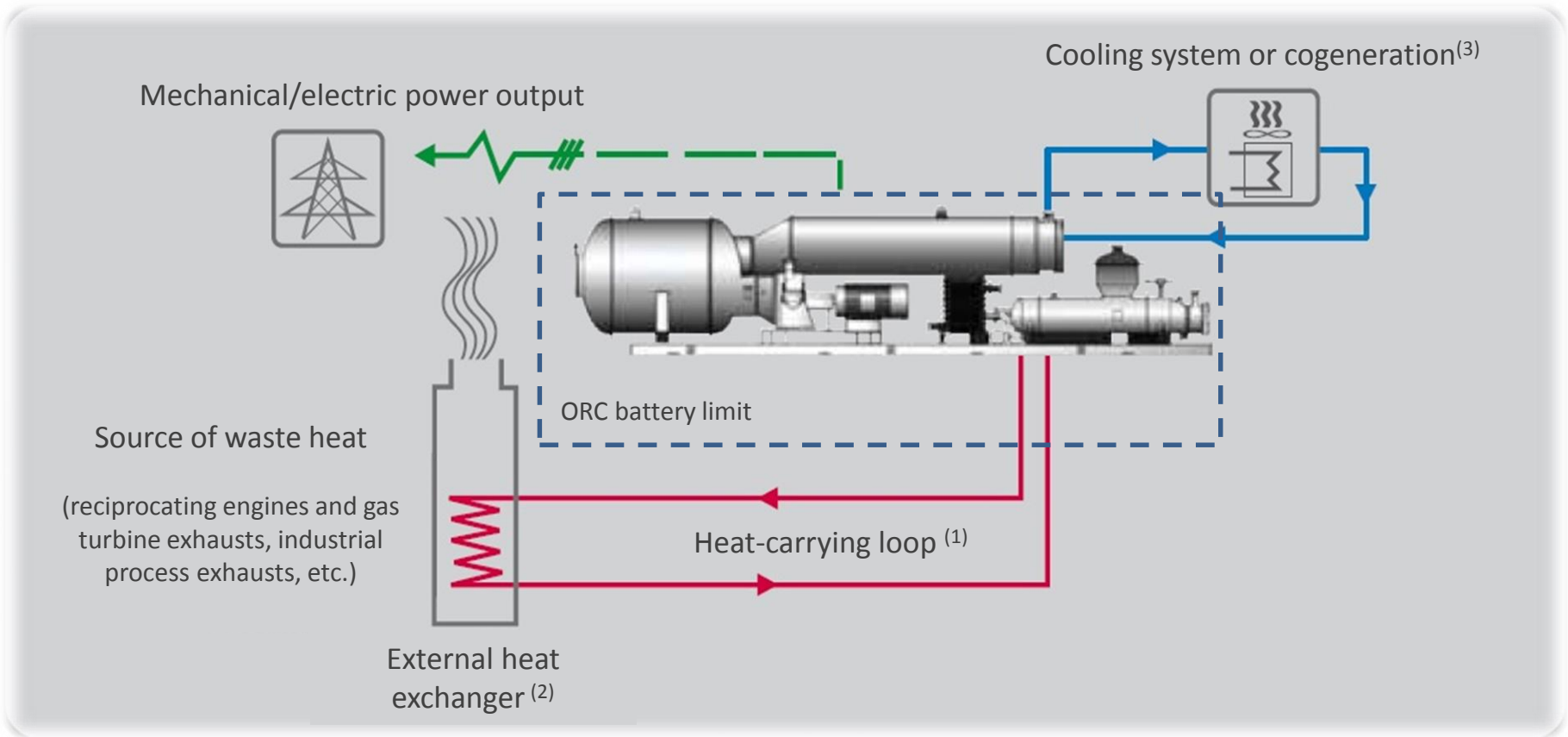
(2) Fast start-stop procedure (ca. 20 min), partial load operation (down to 10% of nominal load)

ORC performance at partial load



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Waste heat to power ORC plant



Note

- 1) Heat-carrying loop may be filled with verse media e.g. thermal oil, saturated steam, pressurized water or it can be replaced by a direct exchange between the exhaust and the organic fluid
- 2) Possibility to exploit multiple thermal sources
- 3) Cooling tower, water cooled condenser, air cooled condenser, other

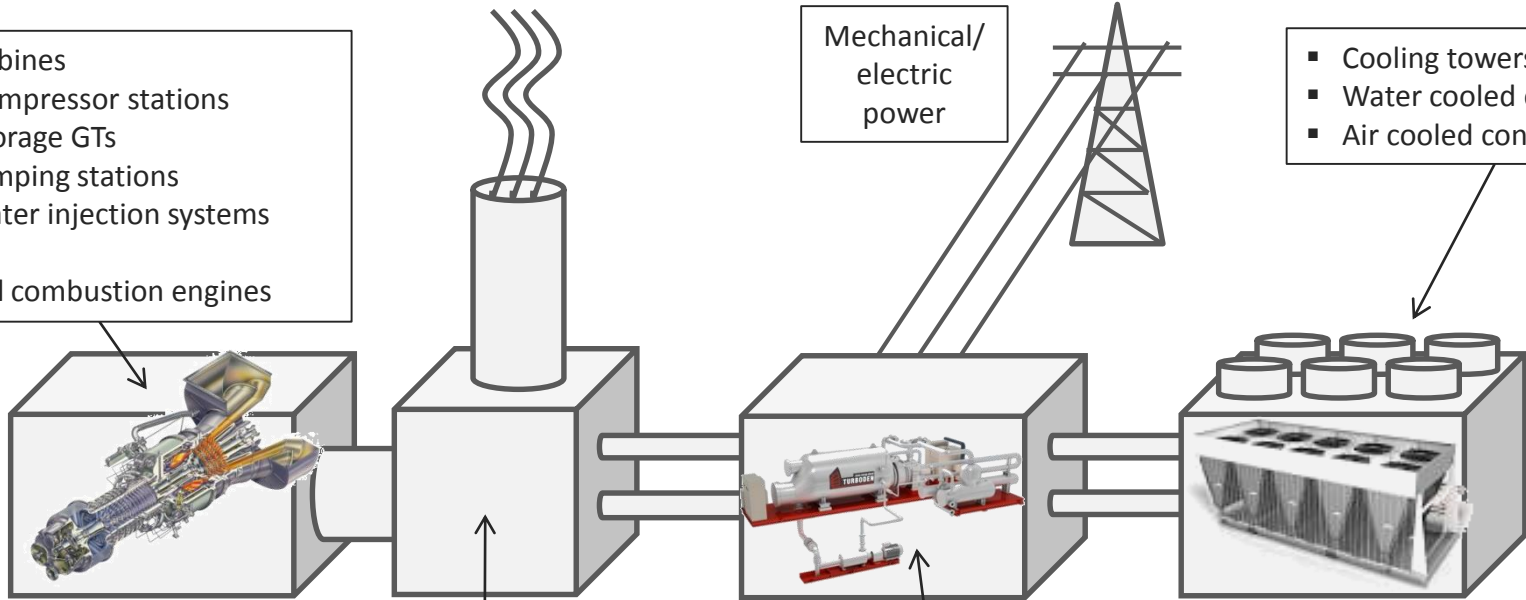
ORC finds many applications in the Oil&Gas sector

- A. Gas turbines exhaust gas
Gas compressor stations, natural gas liquefaction, gas storage, etc.
- B. Hot water from exhausted oil wells
- C. Associated Petroleum Gas (APG)
- D. Refinery hot streams
Distillation columns, Oil/Gasoline/Kerosene production, etc.

Oil&Gas applications

Ⓐ Gas turbines exhaust gas

- Gas turbines
 - Gas compressor stations
 - Gas storage GTs
 - Oil pumping stations
 - Sea water injection systems
 - ...
- Internal combustion engines



- Cooling towers
- Water cooled condensers
- Air cooled condensers

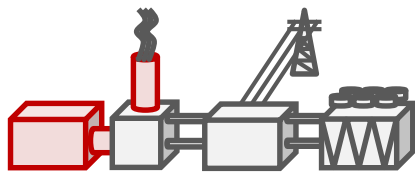
- Heat exchanger**
- Direct exchange
 - Thermal oil

- Organic Rankine Cycle**
25 ÷ 35% additional power ⁽¹⁾

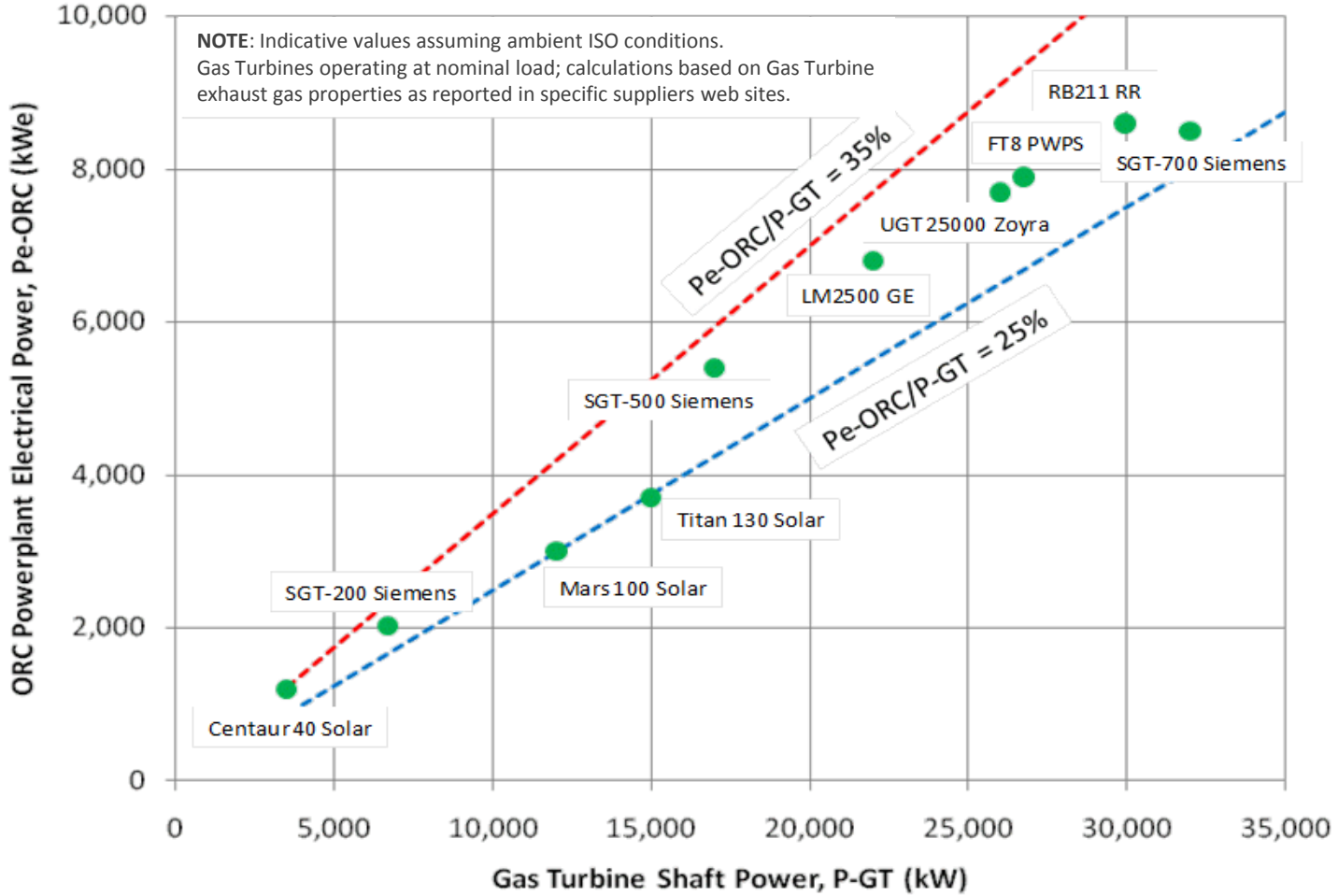
Turboden References

- **TransGas – Canada**
GT power: 3.5 MWe
ORC power: **1 MWe**
Thermal oil circuit
In operation since Q4 2011
- **PolympeX – Russia**
GT power: 25 MWe
ORC power: **3 MWe**
CHP th. power: 15 MWth
Direct Exchange
Expected start up: 2015

(1) Percent of the prime mover nominal power



25% - 35% of the prime mover nominal power output is recovered through ORC



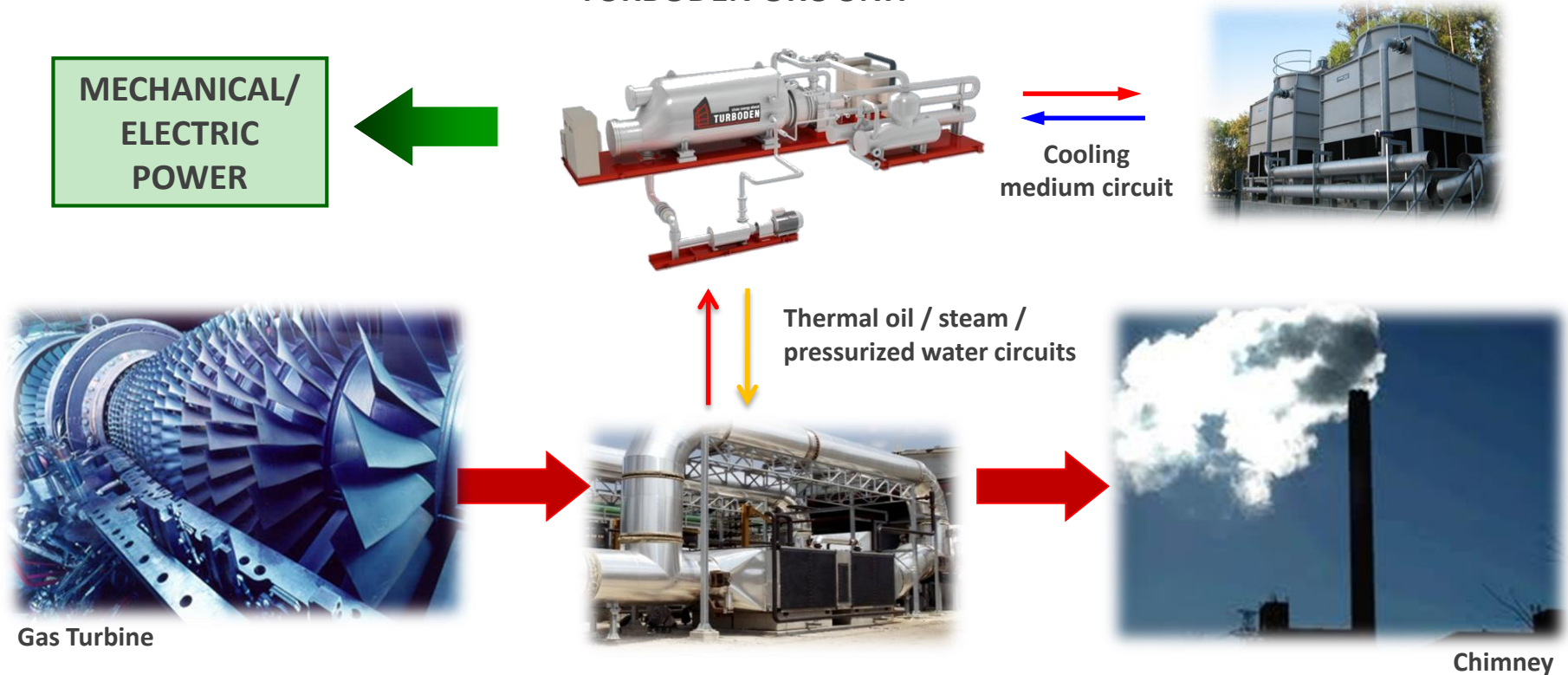
Up to 35% additional power

Thermal oil heat carrier loop

ORC-based heat recovery solution:

- Thermal oil / pressurized water / steam heat recovery exchangers with exhaust gas
- Silicon-based fluids, hydrocarbons or refrigerants used as working fluids
- Water cooled or air cooled condensers employable

TURBODEN ORC UNIT



HEAT EXCHANGERS

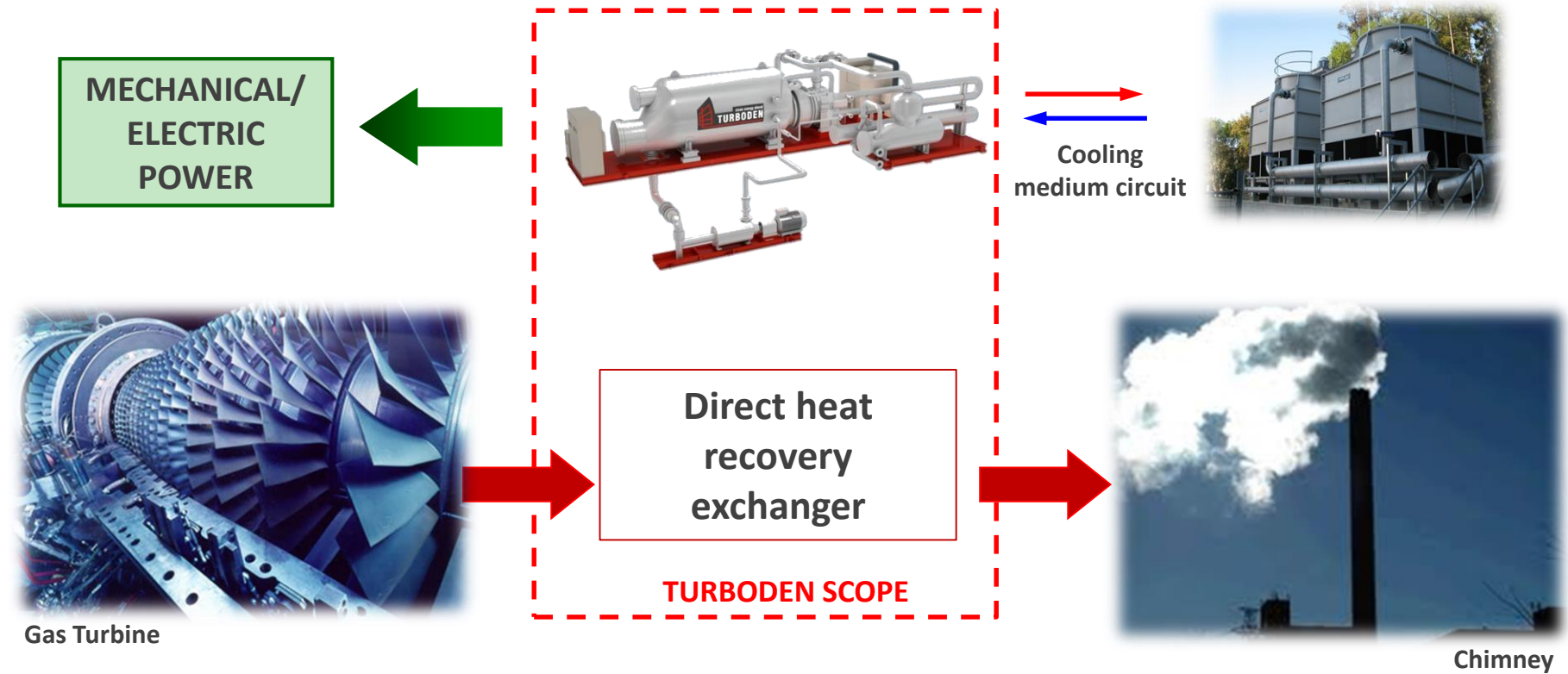
Typically not included in Turboden scope of supply

Direct exchange

ORC direct exchange solution:

The thermal energy contained in the exhaust gas is transferred directly, through direct exchange between exhaust gas and the working fluid, to the ORC plant.

For this solution the primary exchanger (exhaust gas / ORC working fluid) is included in the Turboden scope of supply.

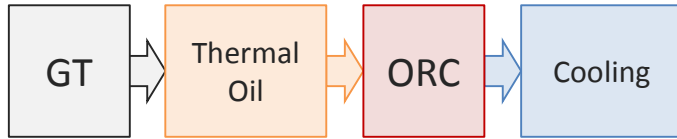


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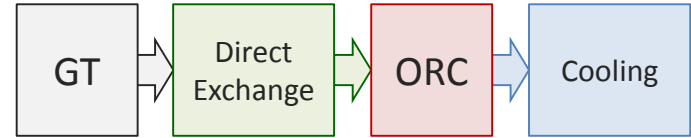
CCGT Schemes

1 turbine, 1 ORC

Thermal Oil

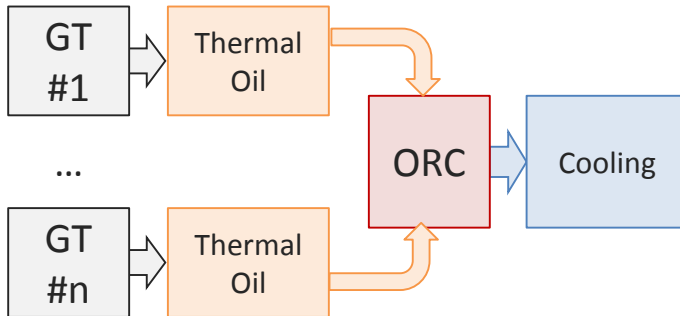


Direct Exchange

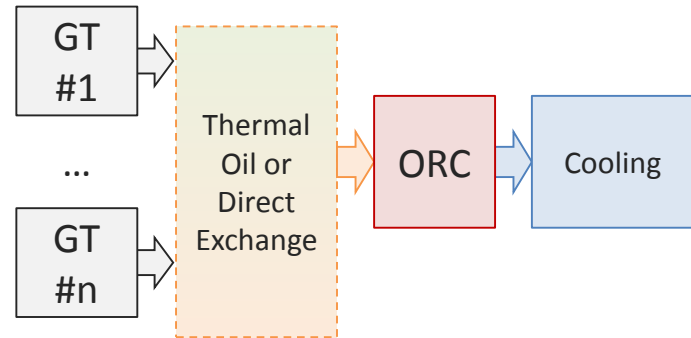


Multiple Heat Sources

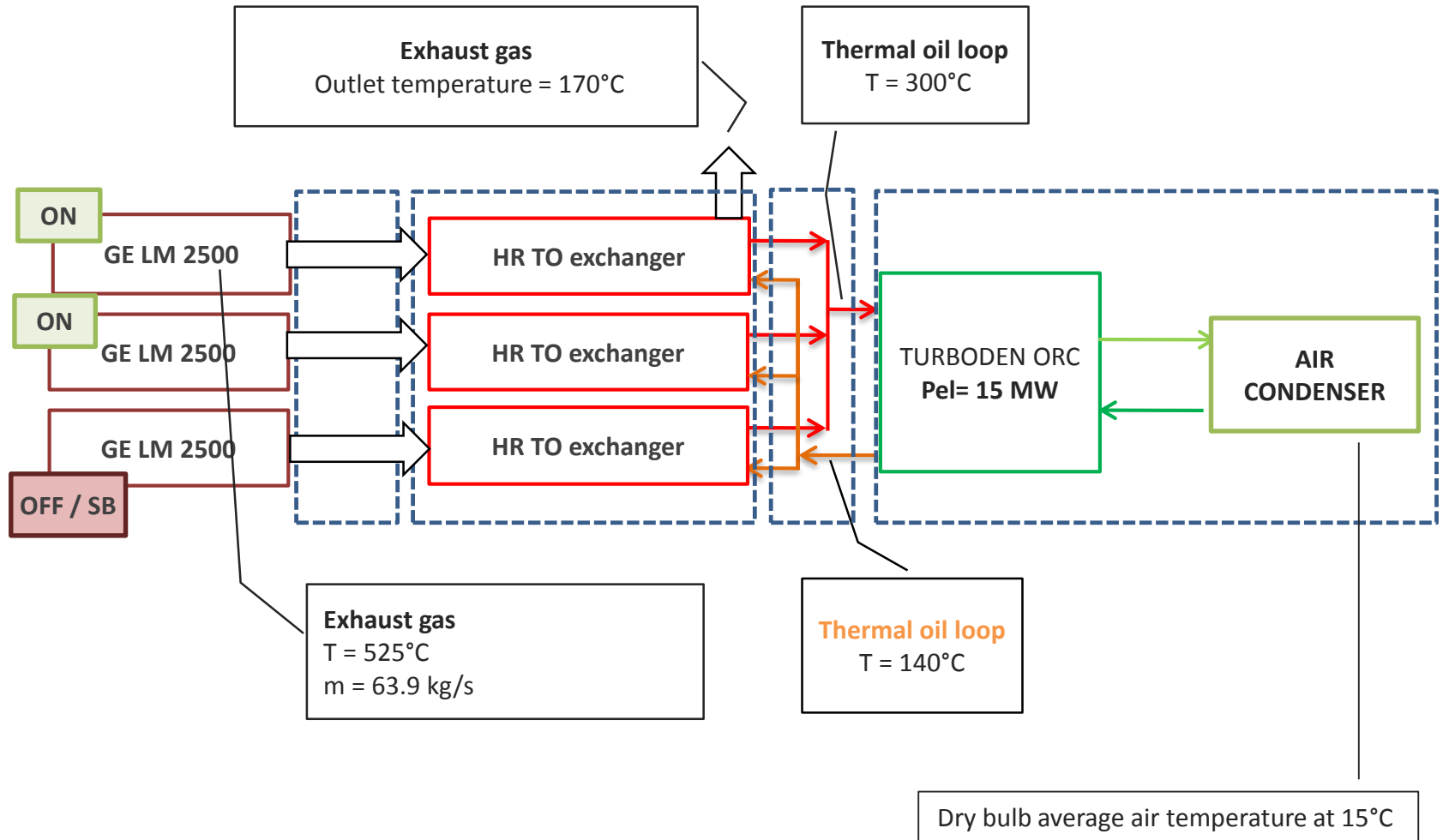
Separated thermal oil heat recovery exchangers



Exhaust gas conveyed to a single heat recovery exchanger



CCGT multiple recovery example



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A huge potential resides in WHR on Oil&Gas infrastructures

Reference case

Germany Gas Transmission System Operator

28 Gas Compressor Stations on 11,550 km network (1)



Capacity factor considered: **45% (2)**

Total mechanical drive installed capacity: **990 MW**



Equivalent power considered: **445 MW**

ORC recovery factor: **30%**

ORC potential: **135 MWe**

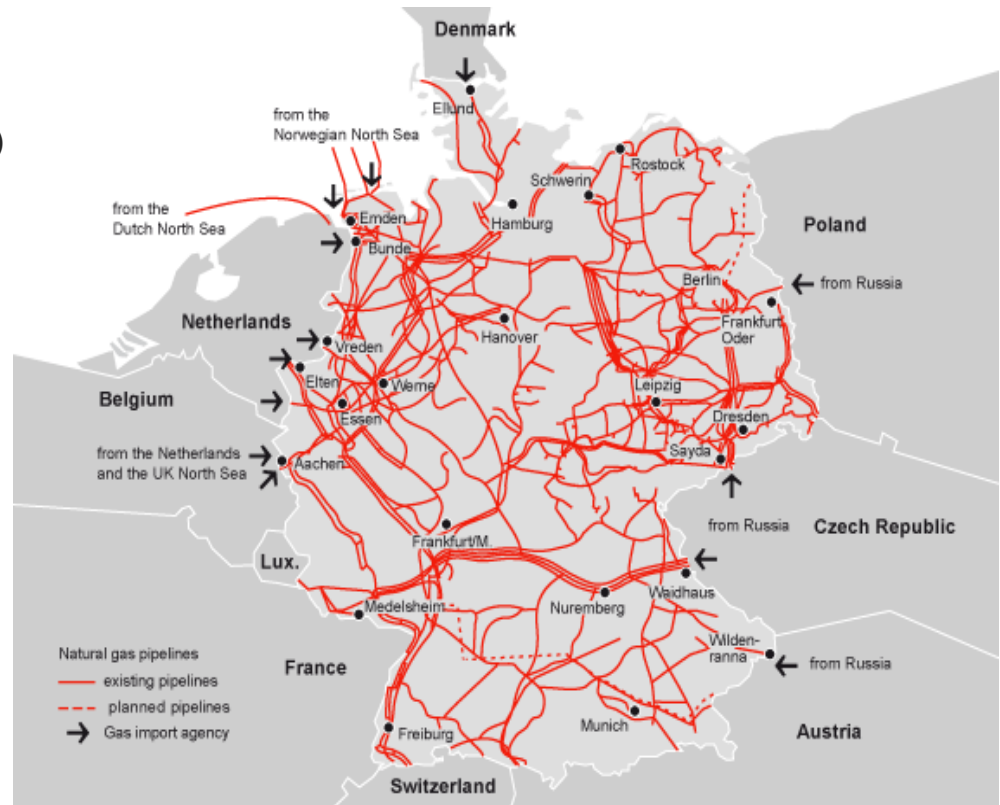


Equivalent operating hours: **6,000 h/y(3)**

Energy savings: **800 GWhe → 48 M€/y(4)**

or 208 million cubic meter of natural gas(5)

Emission avoided: **320,000 t CO₂/y(6)**



Example: Germany midstream application

- (1) Source ENTSOE Ten Year Network Development Plan 2011-2020
- (2) Assuming 3 gas turbine per site. Average power: 1 nominal (100%) + 1 partial load (35%) + 1 backup (0%)
- (3) Assuming seasonal fluctuations in GCS operation, ORC availability > 95%
- (4) Assuming an electricity value of 60 €/MWh
- (5) Assuming a consumption of 260 mc of natural gas per MWh of power generated
- (6) Assuming an average emission factor of EU power generation plants of 400 t CO₂ per GWh (source IEA 2013)

Natural Gas Compressor Stations: a big opportunity for Heat Recovery

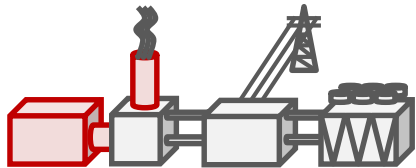
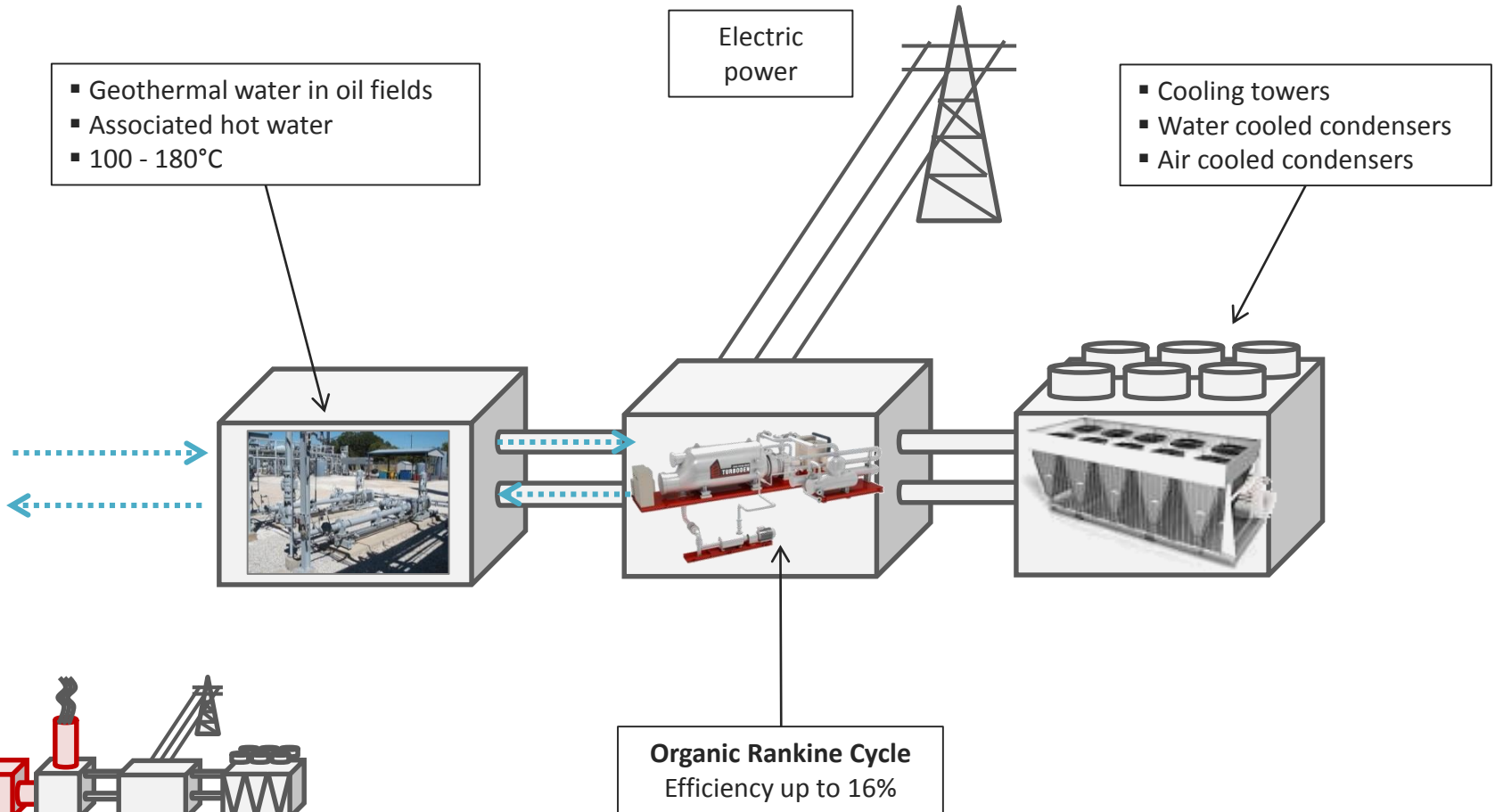
- World natural gas yearly consumption: about 3,000 billion m³
- Compressor stations usually placed at 40 to 100 miles intervals along the pipelines
- Dozens of GW of compression capacity to move natural gas from production sites to users
- Most compressor stations are operating on an open cycle (efficiency about 30-35%)

Dozens of GW of thermal power (in form of hot exhaust gas) are wasted into the atmosphere



Oil&Gas applications

Ⓑ Hot water from exhausted oil wells

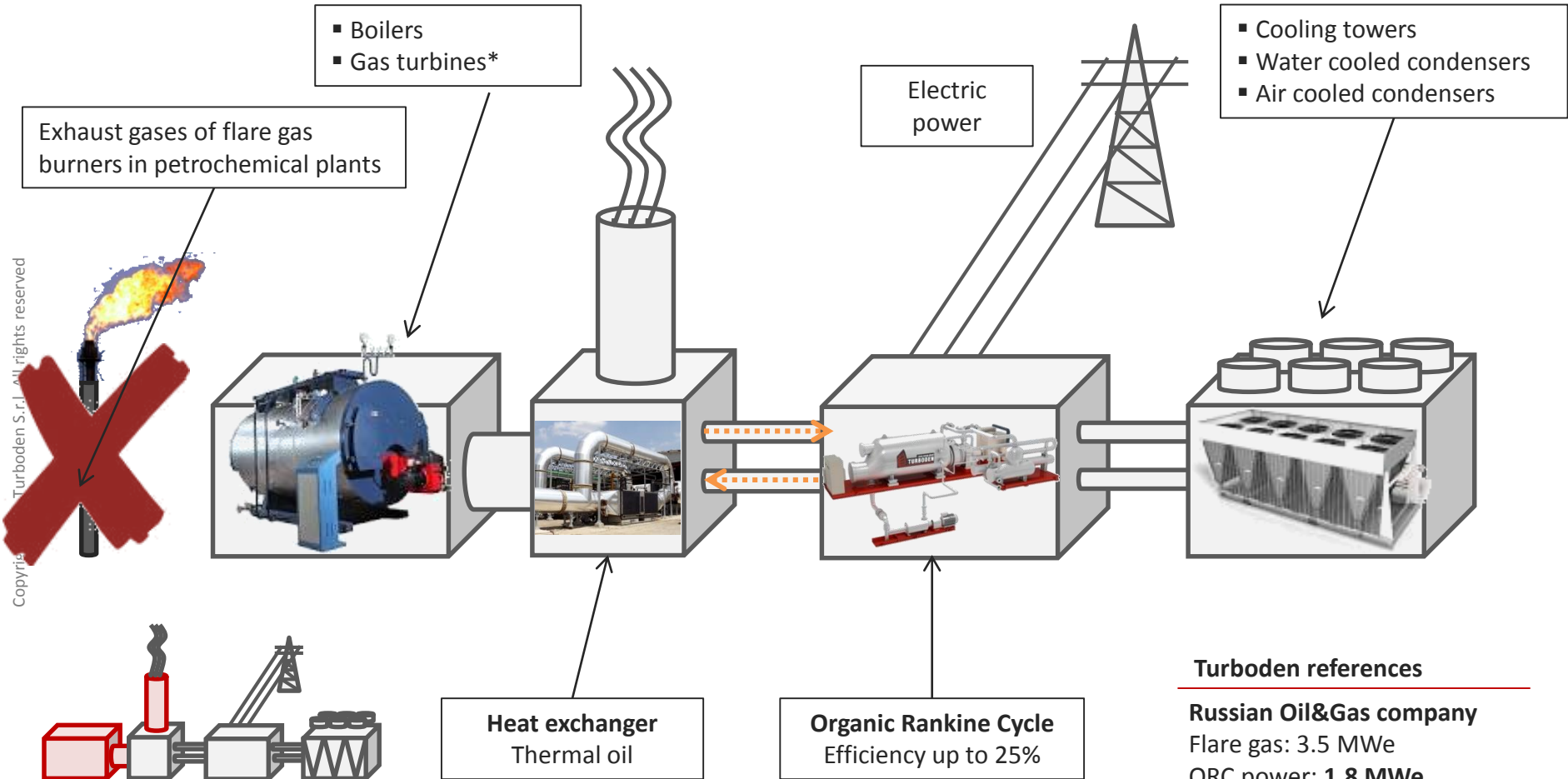


Turboden references

31 MW in 9 geothermal plants

Oil&Gas applications

© Associated Petroleum Gas (APG)



Turboden references

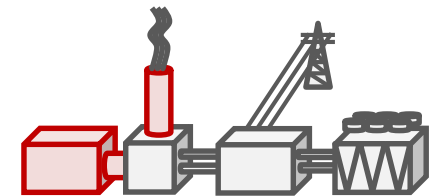
Russian Oil&Gas company

Flare gas: 3.5 MWe

ORC power: **1.8 MWe**

Burner + thermal oil circuit

Start up: Q1 2015



* Low heating value gas turbine

Turboden reference – APG exploitation

Site: Perm, Russia

Customer/End user: LabNT/LUKoil

Status: started up in January 2015

Heat source: flare gas burning (boiler designed to burn gas with a minimum lower calorific value of 4,500 kcal/Nm³)

Heat source temperature: thermal oil at 300 °C

Inlet/Outlet water temperature: 65/95 °C

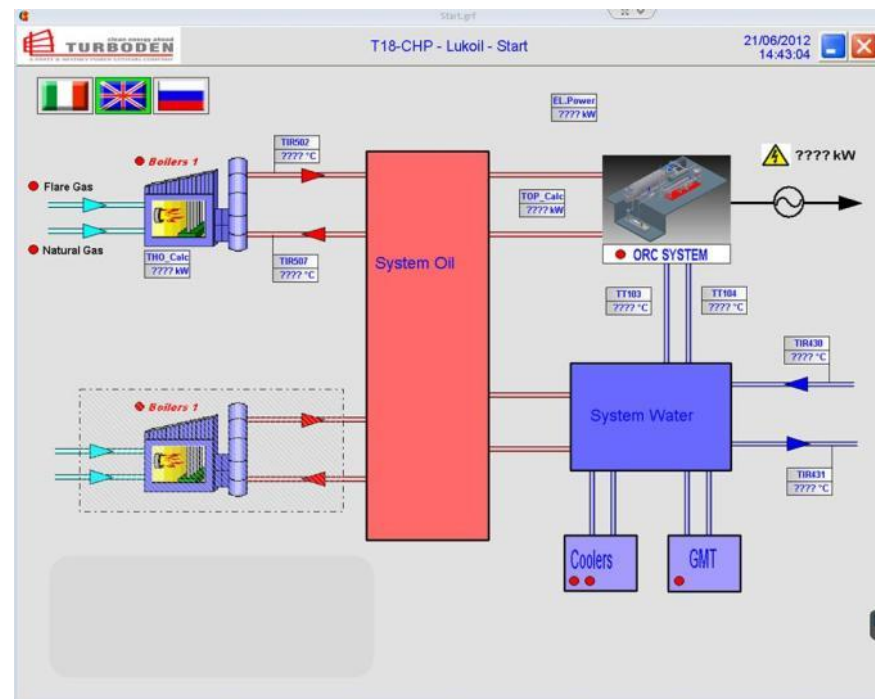
Electric power: ~1.8 MW

Net electric efficiency: ~18%



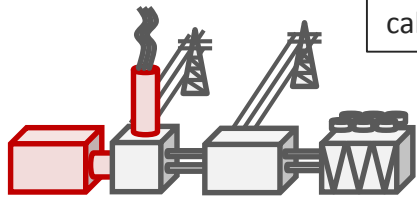
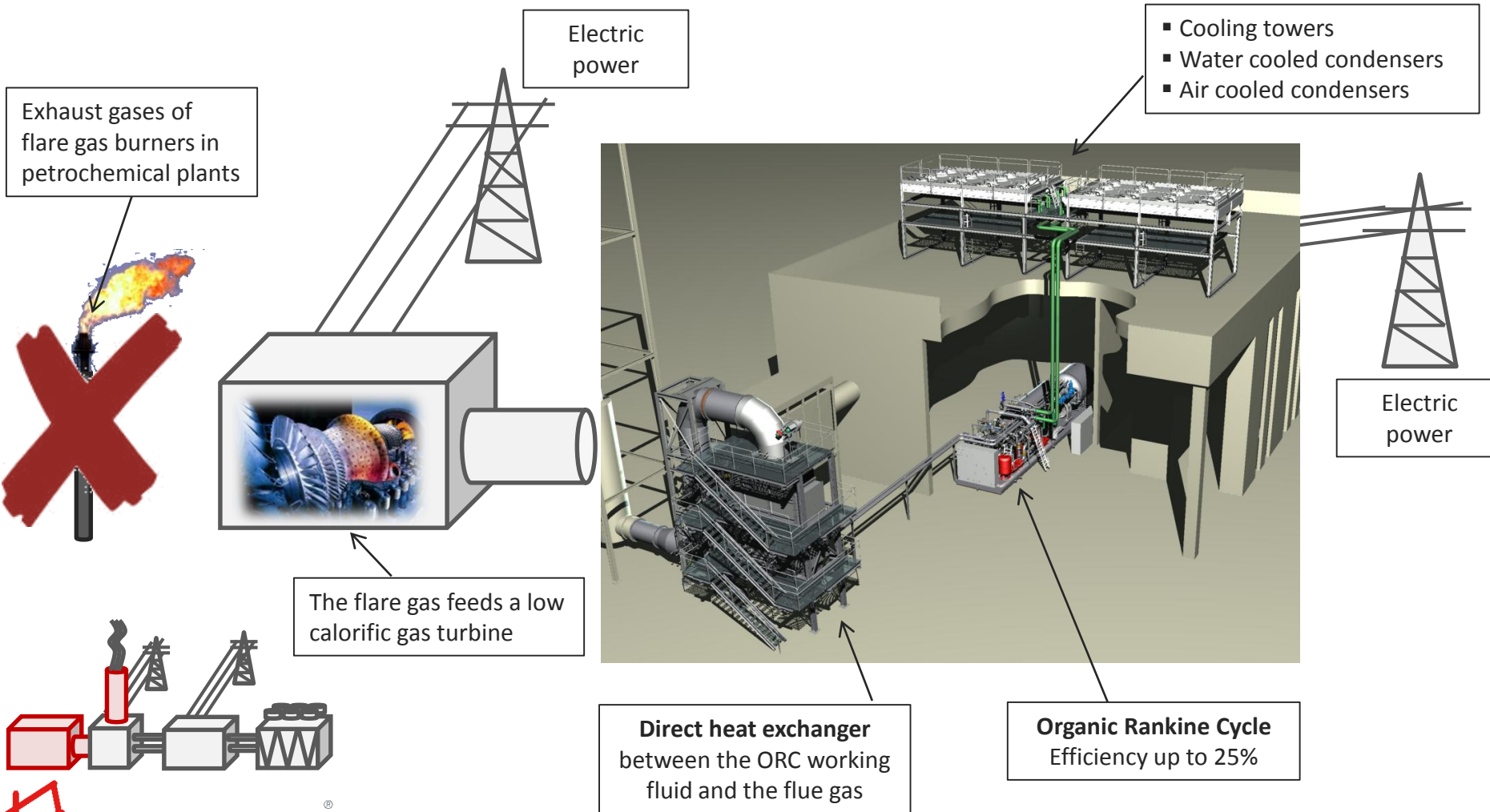
Project description

Flare gas from oil extraction wells is burned to heat up thermal oil which is used to feed up an ORC CHP unit. The electricity produced reduces the plant consumptions, whereas the hot water produced is exploited in oil refinery processes including warming up of refined products to be pumped.



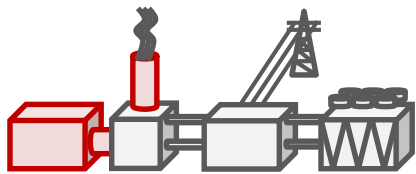
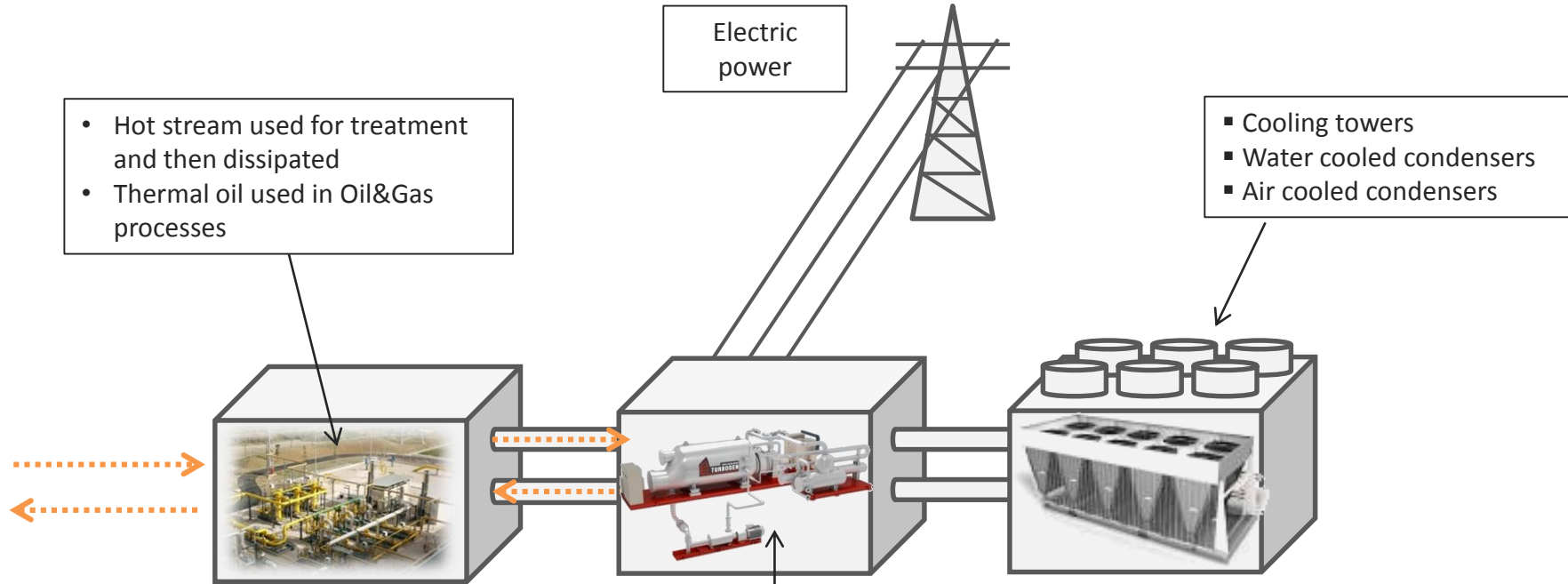
APG exploitation – Further development

As a further development of the solution a low calorific value gas turbine can be employed and the ORC can be configured in direct exchange mode, exploiting the exhaust gases of the gas turbine.



Oil&Gas applications

④ Refinery hot streams



Organic Rankine Cycle
Efficiency up to 25+%⁽¹⁾

Turboden case study

Thermal oil power: 53 MWth
ORC electric power: 10 MW

(1) Heat carrier temperature above 300°C

References – Combined Cycle Gas Turbines



Gas Compressor Station – TransGas

Heat recovery from **Solar CENTAUR** gas turbine in a **Gas compressor station** in **Canada**

Gas turbine prime power: **3.5 MWe**

Gas turbine efficiency: **28%**

ORC electric power: **1 MW**

General contractor: **IST**

Final client: **TransGas**

Start up: **in operation since November 2011**



Gas Compressor Station

Heat recovery from **Solar TITAN 130** gas turbine in a **Gas Turbine Power Plant (GTPP)** in **Russia** (Moscow region)

Gas turbine prime power: **15 MWe**

Gas turbine efficiency: **30%**

ORC electric power: **3 MW** direct exchange cogenerative solution

ORC thermal power: **15 MW** of hot water at **90° C**

General Contractor: **Energo development LCC**

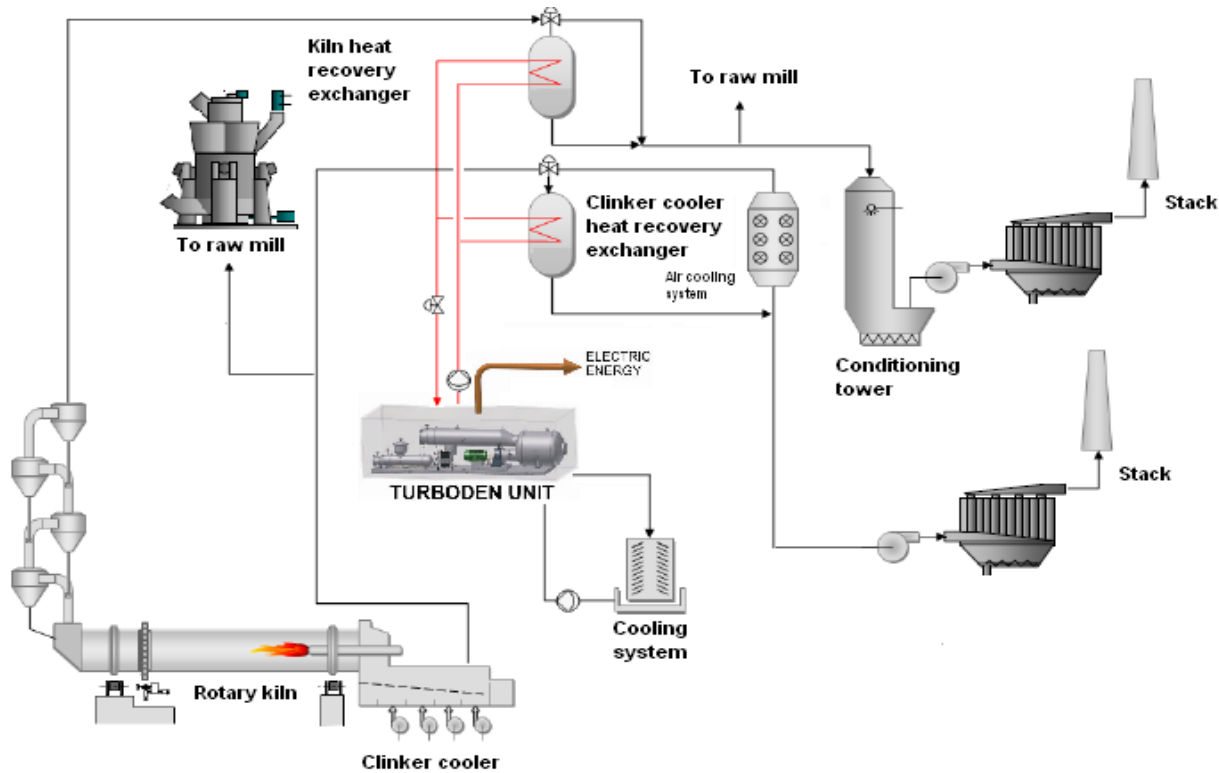
Final Client: **undisclosed**

Status: **under construction**

References – Internal Combustion Engines

Project	ORC Module	Site	Engines
Pisticci I	18 HR SPLIT (1.8 MWe) Start up: Q2 2010	Pisticci (IT)	3 x 8 MWe Wartsila Diesel engines
Termoindustriale	6 HR SPLIT (0.6 MWe) Start up: Q4 2008	Pavia (IT)	1 x 8 MWe MAN Diesel engine
Pisticci II	40 HR SPLIT (4 MWe) Start up: Q2 2012	Pisticci Scalo (IT)	2 x 17 MWe Wartsila Diesel engines
Cereal Docks	6 HR DIR. EXCH. (0.6 MWe) Start up: Q1 2012	Portogruaro (IT)	1 x 7 MWe Wartsila Diesel engine
E&S Energy	6 HR SPLIT (0.6 MWe) Start up: Q2 2010	Catania (IT)	2 x 1 MWe JGS/GE gas engines + 3 x 0.8 MWe JGS/GE gas engines + 1 x 0.6 MWe JGS/GE gas engine
Ulm	10 HR cogenerative (1 MWe) Start up: Q3 2012	Senden (DE)	2 x 2 JGS/GE gas engines (+ additional heat from the process)
Kempen	6 HR cogenerative (0.6 MWe) Start up: Q1 2012	Kempen (DE)	Gas engines
Mondopower	10 HR (1 MWe) Start up: Q4 2012	Chivasso (IT)	1 x 17 MWe Wartsila Diesel engine
HSY	14 HR (1.3 MWe) Start up: Q3 2011	Ämmässuo, Espoo (FIN)	4 x 4 MWe MWM gas engines
Fater	7 HR DIR. EXCH. (0.7 MWe) Start up: Q2 2013	Pescara (IT)	1 x 8 MWe Wartsila Diesel engine

WHTP with ORC in cement industry



Start up year	References in cement plants	Heat source	ORC gross electric power [MW]
2010	Italcementi - Ciment du Maroc, Marocco	PH + CSP	1.5
2012	Holcim Romania	PH + CC	4
2014	Holcim Slovakia	PH + CC	5
Under construction	Heidelberg Cement – Cartpatcement Romania	PH + CC	4

References – Steel industry

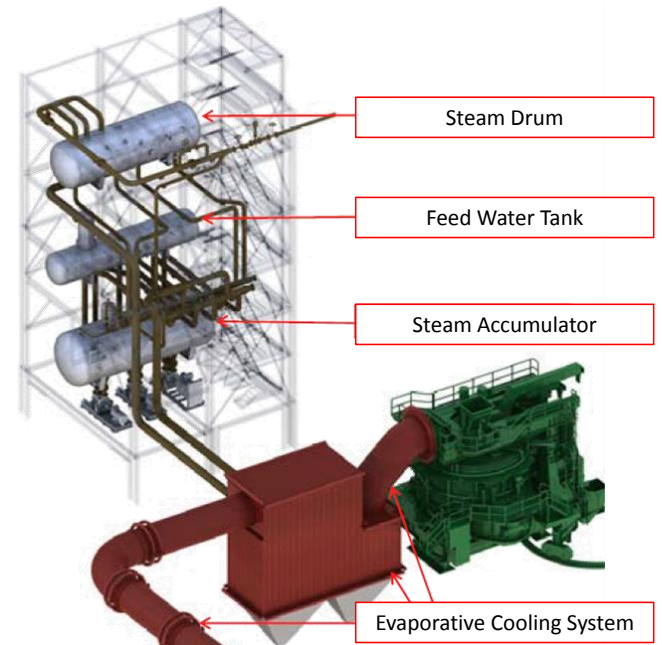
1. Electric Arc Furnace Data

Heat source	EAFF process off-gas
Steel production	1M tons/year
Heats per day (average)	32
EAFF hourly production	133 tons/hour
Tapping weight	100 tons
Tapping temperature	1,600°C
Charge weight	113 tons
Average off-gas temperature (core temperature ex EAFF)	1,100°C
Average off-gas flow rate	100,000 – 140,000 Nm ³ /h



2. Heat recovery system Data

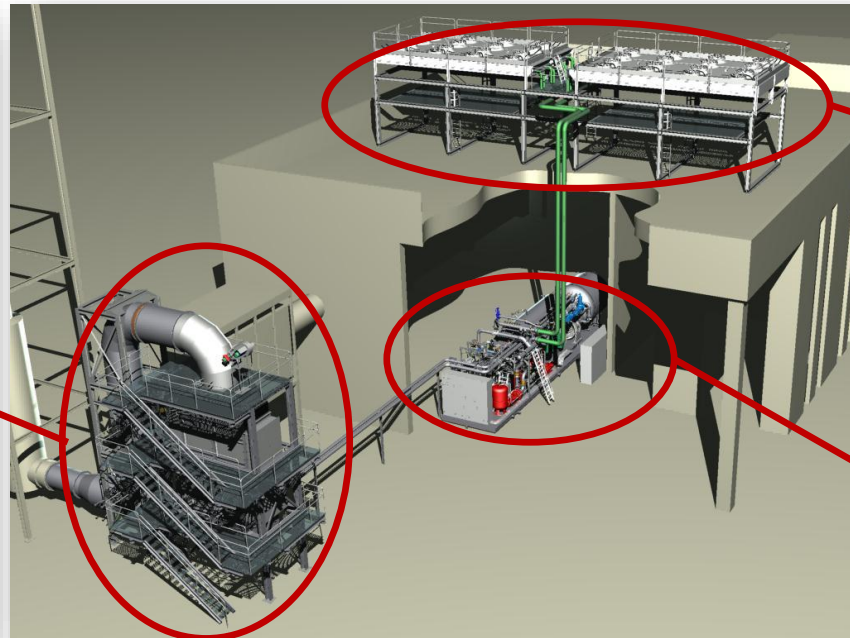
Nominal steam data at steam drum	247°C – 38 bar(a)
Water content cooling system (pipes + tank)	approx. 37 m ³
Capacity of steam accumulation of cooling system	1,442 kg
Inlet thermal power to the ORC	13,517 kW
Steam temperature into ORC	228÷245°C
Condensate temperature out from ORC	100°C
Thermal power to the cooling water	10,640 kW
Cooling water temperatures (in/out ORC)	26°C / 44°C
Gross electric power output	2,680 kW
Net electric power output	2,560 kW



In operation since December 2013

References – Direct Exchange

Project	Customer	Site	Heat Source	ORC power	Start up
Cereal Docks	food industry	Italy	7 MW diesel engine	700 kW	Q1 2012
Alma CIS/Fater	personal care industry	Italy	18V32 diesel engine	700 kW	Q2 2013
Natsteel/TATA group	steel industry	Singapore	hot rolling mill reheating furnace	700 kW	Q1 2013
Polympex	cogenerative	Russia	Solar TITAN 130 gas turbine	3,000 kW	Q4 2014



References – Biomass

218 references in operation, 248 MW installed

Example of large size ORC (thermal oil input): West Fraser Mills Ltd

Site 1: Chetwynd, British Columbia - Canada - 2 x Turboden ORC Units; expected start up Q2 2015

Site 2: Fraser Lake, British Columbia - Canada - 2 x Turboden ORC Units; in operation since November 2014

- Fuel:** sawmill scraps, woodchips
- ORC heat carries:** thermal oil
- ORC electric power:** 4 x 6.5 MW



West Fraser



a group company of  MITSUBISHI HEAVY INDUSTRIES, LTD.

References – Geothermal

6 references in operation, 19 MW installed



Sauerlach

Customer: SWM - StadtWerke München (Munich multi-utility)

Site: Sauerlach, Germany

ORC size: 5.6 MWe + 4 MWth to district heating

Start up: January 2013

Scope of supply: Complete ORC supply, air condenser includes



Dürrnhaar

Customer: Hochtief Energy Management GmbH

Site: Dürrnhaar (München), Germany

ORC size: 5.6 MWe

Start up: December 2012

Scope of supply: full EPC for ORC, air condenser and BOP



Kirchstockach

Customer: Hochtief Energy Management GmbH

Site: Kirchstockach (München), Germany

ORC size: 5 MWe

Start up: January 2013

Scope of supply: full EPC for ORC, air condenser and BOP

Key factors:

- **Close to the Munich urban area (<10 km)**
- Coupled with urban **district heating network**
- Possibility to work **on island mode** (Sauerlach)
- **Medium enthalpy: 140°C**
- **Non-flammable** working fluid
- **Small area covered** due to houses in the nearby
- **Turboden supply all the components** except geothermal pump and circuit

References – Low temperature water



Waste to energy – Mirom, Belgium

Heat recovery from **pressurized water boiler** in **waste incinerator**

Customer: MIROM

Location: Roeselare, Belgium

Source: hot water at 180° C (back at 140° C)

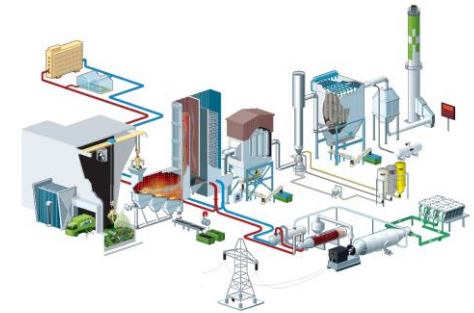
Cooling source: air coolers

ORC electric power: 3 MW

Electrical efficiency: 16.5%

Availability: >98%

Start up: Q2 2008



Waste to energy – Séché, Francia

Heat recovery from **pressurized water boiler** in **waste incinerator**

Customer: Séché Environnement Usine - Alcea

Location: Nantes, France

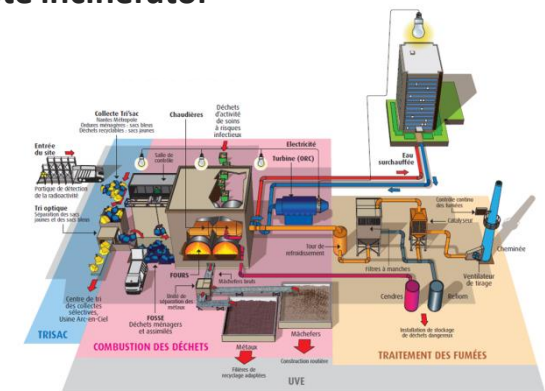
Source: hot water at 200° C (back at 130° C)

Cooling source: air coolers

ORC electric power: 2.4 MW

Electrical efficiency: 16.5%

Start up: Q3 2014



Case study – Heat recovery from GTs (1/2)

Reference case

3 gas turbine (Siemens SGT 600) at full load (100%)

Site: Middle East



Cooling temperature (ambient air): 26°C

Cooling system: Air cooled condenser



ORC Size: 15 MWe

Net Power: 13.2 MWe

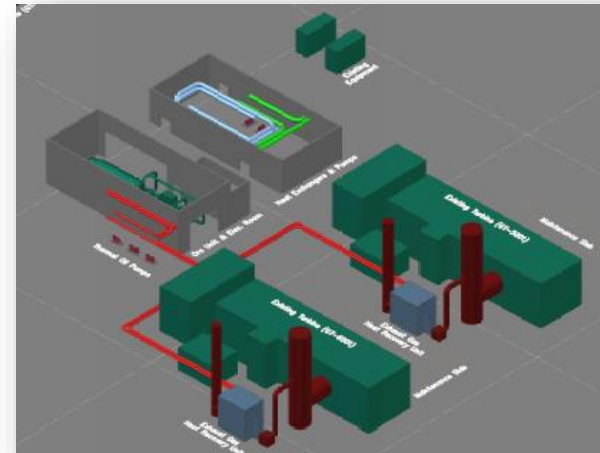


Equivalent operating hours: 8,000 h/y ⁽¹⁾

Energy savings: 106 GWh → 6.3 M€/y ⁽²⁾

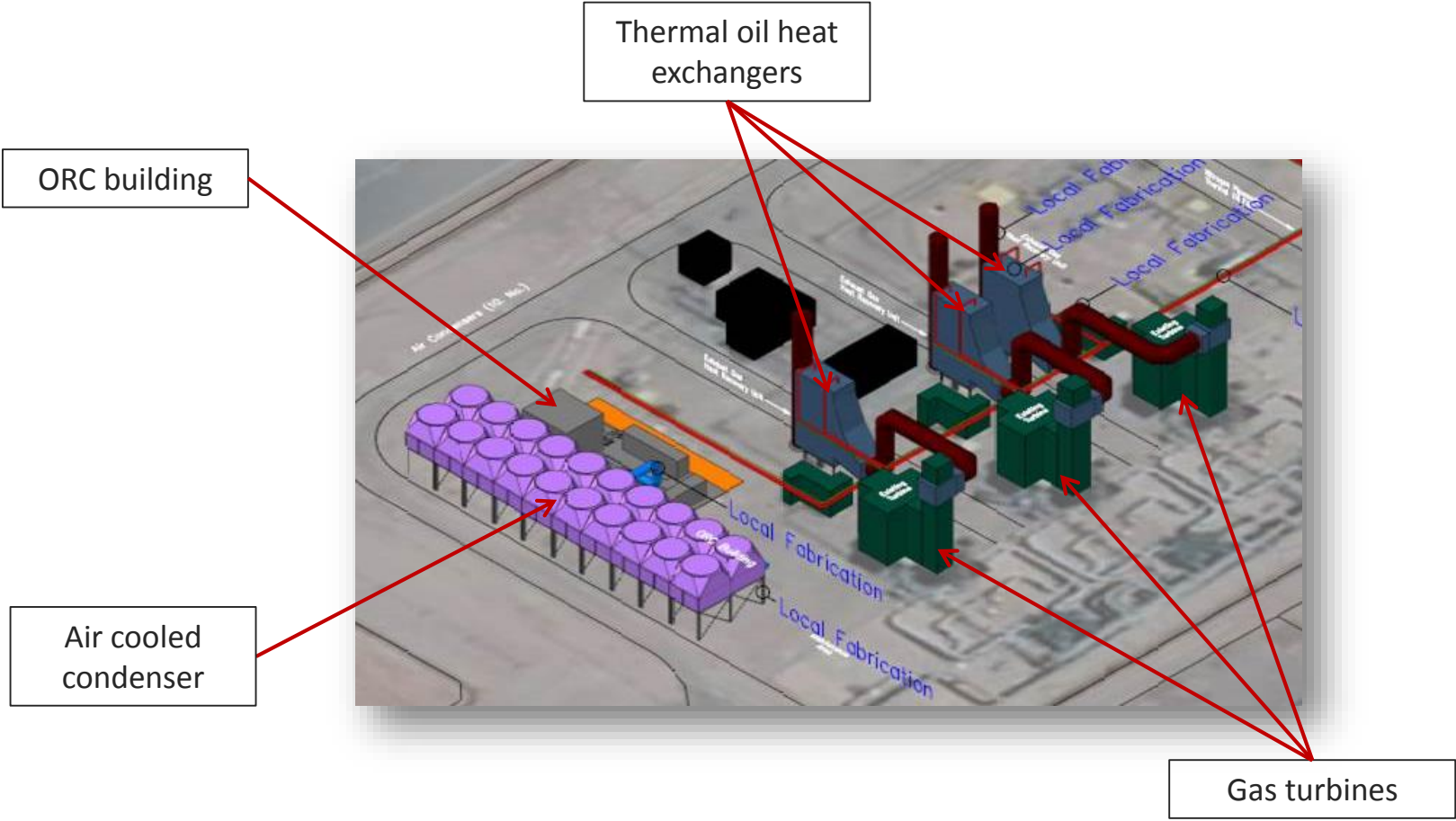
or 22 mcm of natural gas ⁽³⁾

Emission avoided: 68,900 t CO₂/y ⁽⁴⁾



- (1) Middle East Gas infrastructures work continuously, ORC availability > 95%
- (2) Assuming an electricity value of 80 €/MWh as barrels of oil savings
- (3) Assuming a consumption of 260 mc of natural gas per MWh of power generated
- (4) Assuming an average emission factor of EU power generation plants of 650 t CO₂ per GWh (source IEA 2013)

Case study – Heat recovery from GTs (2/2)



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Conclusion

- ORC technology is a proven way to reduce equivalent fuel consumption and CO₂ emission in Oil&Gas infrastructure equipment
- Turboden ORC technology for waste heat recovery makes Oil&Gas greener
- Turboden has experience in Oil&Gas technical codes and standards
- Provide us with your waste heat data to allow us studying an optimum heat recovery solution

Turboden at a Glance



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Turboden strong points



- Participation in national & EU research programs
 - Cooperation with EU Universities and Research Centres
 - Thermodynamic cycle optimization
 - Working fluid selection & testing
 - Thermo-fluid-dynamic design and validation
 - Implementation & testing of control/supervision software
 - Many patents obtained
- Pre-feasibility studies: evaluation of technical & economical feasibility of ORC power plants
 - Customized proposals to maximize economic & environmental targets
- Complete in-house mechanical design
 - Proprietary design and own manufacturing of ORC optimized turbine
 - Tools
 - ☞ Thermo-fluid-dynamic programs
 - ☞ FEA
 - ☞ 3D CAD-CAM
 - ☞ Vibration analysis
- Outsourced components from highly qualified suppliers
 - Quality assurance & project management
 - In-house skid mounting to minimize site activities
- Start-up and commissioning
 - Maintenance, technical assistance to operation and spare parts service
 - Remote monitoring & optimization of plant operation

Notes

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Notes

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